

Case Studies





Applying basic principles to real world situations: Case Studies



Case study presentation agenda

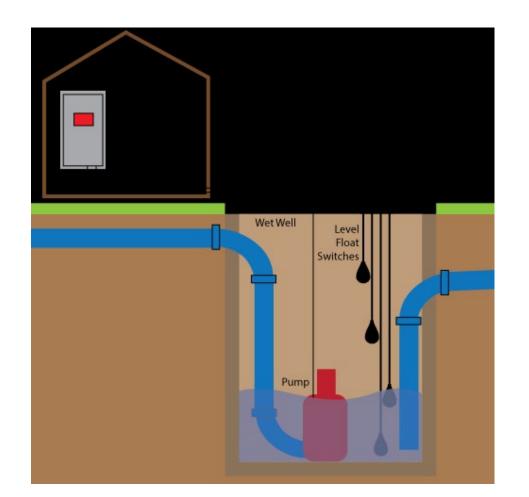
- Describe "as found" case study configurations
- Look for indicators of energy reduction opportunities (prescreening method)
- Apply the MEASUR software to as found data
- Describe changes made to improve the situation





Welches Point Wastewater Lift Station (Milford, Connecticut)

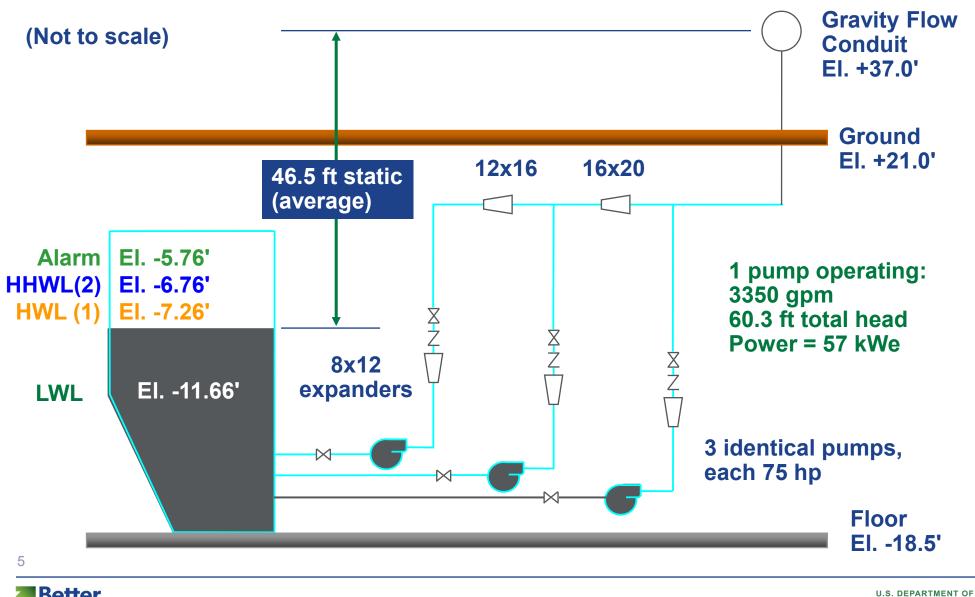
- This case study is based on work done by ITT Flygt for the town of Milford
- Case study technical contact:
- Gunnar Hovstadius (retired)
- Tel: 203-227-4503 or 203-434-4840
- Email:gunnarh@msn.com
- Case study prepared by Don Casada, Diagnostic Solutions







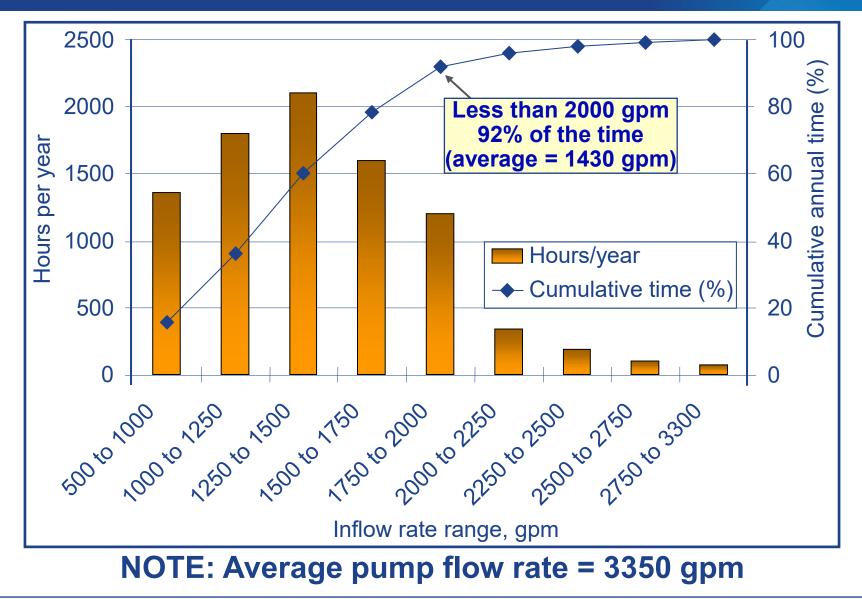
The Welches Point Lift Station cycles pump(s) on/off (run 43% of time) to control wet well level



FNFRG



The pump design capability greatly exceeds the normal operational requirement







Putting the box around the pump and motor for the existing flow and head condition

rpm

°F

cSt

hp

rpm

/kWh

gpm

V

kW

V

| Pump Type | End Suction Sewage | |
|-------------------------|---------------------|-----|
| Pump Speed | 1170 | |
| Drive | Direct Drive | |
| Fluid Type | Water | |
| Fluid Temperature | 68 | |
| Specific Gravity | 1 | |
| Kinematic Viscosity | 1 | |
| Stages | - + 1 | |
| Line Frequency | 60 Hz | |
| Rated Motor Power | 75 | |
| Motor RPM | 1170 | |
| Efficiency Class | Standard Efficiency | |
| Rated Voltage | 460 | |
| Full-Load Amps | 92.3 | |
| Estimate Full-Load Amps | | |
| Operating Hours | 3740 | h |
| Electricity Cost | 0.082 | \$/ |
| Flow Rate | 3350 | (|
| Head | 60.3 | |
| Calculate Head | | |
| Load Estimation Method | Power | |

57

460



Measured Voltage

Motor Power

Putting the box around the pump and motor for the existing flow and head condition

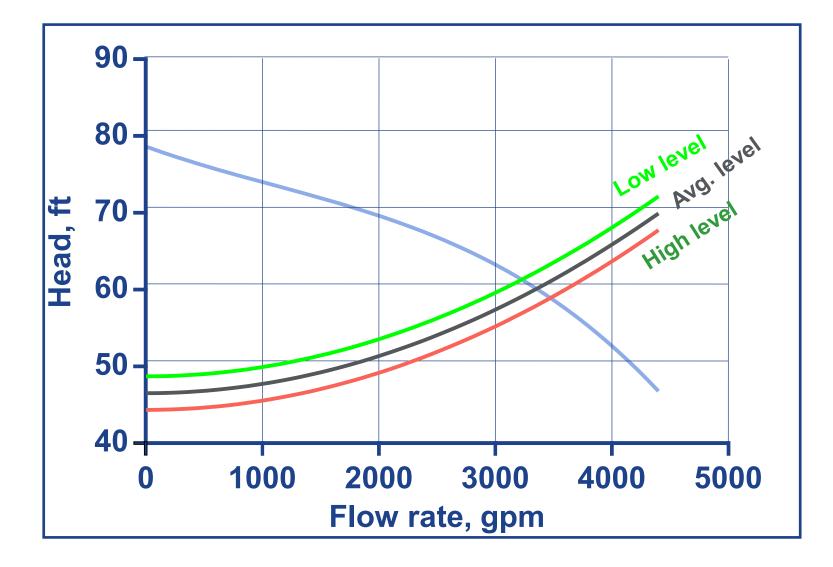
| RESULTS | SANKEY | HELP |
|-----------------------------|----------|--------------|
| | Baseline | Optimal Pump |
| Percent Savings (%) | | 14.0% |
| Pump efficiency (%) | 72.3 | 83.5 |
| Motor rated power (hp) | 75 | 75 |
| Motor shaft power (hp) | 70.6 | 61.1 |
| Pump shaft power (hp) | 70.6 | 61.1 |
| Motor efficiency (%) | 92.4 | 92.4 |
| Motor power factor (%) | 82.2 | 81.2 |
| Percent Loaded (%) | 94 | 81 |
| Drive efficiency (%) | 100 | 100 |
| Motor current (amps) | 87 | 76 |
| Motor power (kW) | 57 | 49.3 |
| Annual Energy (MWh) | 213 | 184 |
| Annual Energy Savings (MWh) | - | 29 |
| Annual Cost | \$17,481 | \$15,119 |
| Annual Savings | - | \$2,361 |

Optimization Rating = (72.3/83.5)100 = 86.6% Existing equipment is not bad





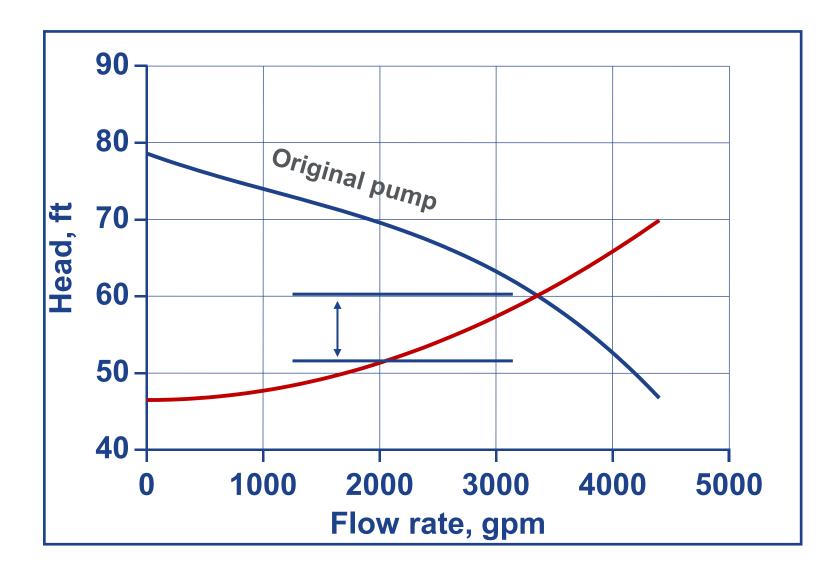
Existing pump & system head-capacity curves







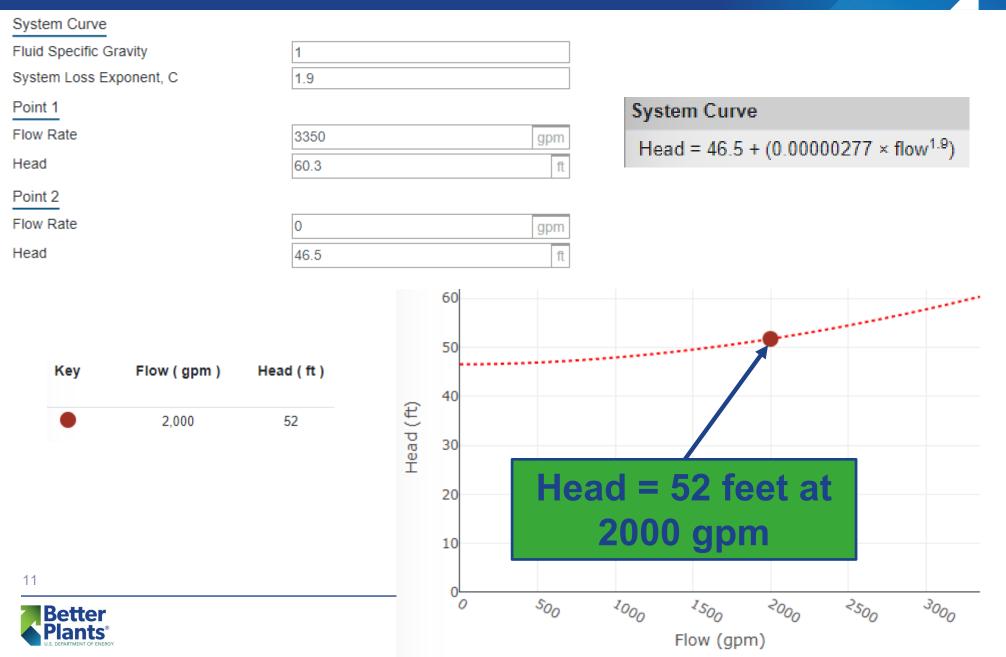
Excessive frictional head losses occur when higher than necessary flow rates occur







The system curve tool can be used to determine head at alternate flow rates



The station processes 752 million gallons/yr; What if we pumped at lower flow rates?

Average running flow rate hours and associated head

| Flow rate | Hours/year | Run fraction | Head (ft) |
|-----------|------------|---------------------|-----------|
| 3350 gpm | 3741 | 0.427 | 60.3 |
| 2500 gpm | 5013 | 0.572 | 54.4 |
| 2000 gpm | 6267 | 0.715 | 51.7 |
| 1500 gpm | 8356 | 0.954 | 49.5 |





Optimized pump at 2500 gpm

| Operating Hours | 5013 |
|------------------------|-------|
| Electricity Cost | 0.082 |
| Flow Rate | 2500 |
| Head | 54.4 |
| Calculate Head | |
| Load Estimation Method | Power |
| Motor Power | 33.97 |
| Measured Voltage | 460 |

| 5013 | hrs/yr |
|-------|--------|
| 0.082 | \$/kWh |
| 2500 | gpm |
| 54.4 | ft |
| Power | ~ |
| 33.97 | kW |
| 460 | V |

Savings = \$17,481 - \$13,964 = \$3,517/yr

| RESULTS | S A | NKEY | HELP |
|-----------------------------|----------|------|---------------------------|
| | Baseline | Op | otimized Pump at 2500 gpm |
| Percent Savings (%) | | | |
| Pump efficiency (%) | 82 | 82 | |
| Motor rated power (hp) | 75 | 75 | |
| Motor shaft power (hp) | 41.9 | 41 | .9 |
| Pump shaft power (hp) | 41.9 | 41 | .9 |
| Motor efficiency (%) | 91.9 | 91 | .9 |
| Motor power factor (%) | 74.7 | 74 | .7 |
| Percent Loaded (%) | 56 | 56 | |
| Drive efficiency (%) | 100 | 10 | 0 |
| Motor current (amps) | 57 | 57 | |
| Motor power (kW) | 34 | 34 | |
| Annual Energy (MWh) | 170 | 17 | 0 |
| Annual Energy Savings (MWh) | — | | |
| Annual Cost | \$13,964 | \$1 | 3,964 |
| Annual Savings | _ | \$0 | 0 |



Optimized pump at 2000 gpm

| - | | |
|--------|----------|-------|
| \cap | porating | Loure |
| | perating | nouis |
| _ | | |

Electricity Cost

Flow Rate

Head

Calculate Head

Load Estimation Method

Motor Power

Measured Voltage

| 6267 | hrs/yr |
|---------|--------|
| 0.082 | \$/kWh |
| 2000 | gpm |
| 51.7 | ft |
| | |
| Power | ~ |
| 26.1765 | kW |
| 460 | V |
| | |

Savings = \$17,481 - \$13,452 = \$4,029/yr

| RESULTS | | SANKEY | HELP |
|-----------------------------|----------|--------|----------------------------|
| | Baseline | (| Optimized Pump at 2000 gpm |
| Percent Savings (%) | | | |
| Pump efficiency (%) | 82 | 1 | 82 |
| Motor rated power (hp) | 75 | | 75 |
| Motor shaft power (hp) | 31.8 | : | 31.8 |
| Pump shaft power (hp) | 31.8 | : | 31.8 |
| Motor efficiency (%) | 90.7 | ! | 90.7 |
| Motor power factor (%) | 67.5 | (| 67.5 |
| Percent Loaded (%) | 42 | | 42 |
| Drive efficiency (%) | 100 | | 100 |
| Motor current (amps) | 49 | | 49 |
| Motor power (kW) | 26.2 | : | 26.2 |
| Annual Energy (MWh) | 164 | | 164 |
| Annual Energy Savings (MWh) | _ | | |
| Annual Cost | \$13,452 | : | \$13,452 |
| Annual Savings | _ | : | \$00 |



Optimized pump at 1500 gpm

| - | | |
|-----|----------|----------|
| () | perating | Houre |
| | Deraunu | 1 IUUI 3 |
| | | |

Electricity Cost

Flow Rate

Head

Calculate Head

Load Estimation Method

Motor Power

Measured Voltage

| 8356 | hrs/yr |
|--------|--------|
| 0.082 | \$/kWh |
| 1500 | gpm |
| 49.5 | ft |
| | |
| Power | ~ |
| 19.315 | kW |
| 460 | V |
| | |

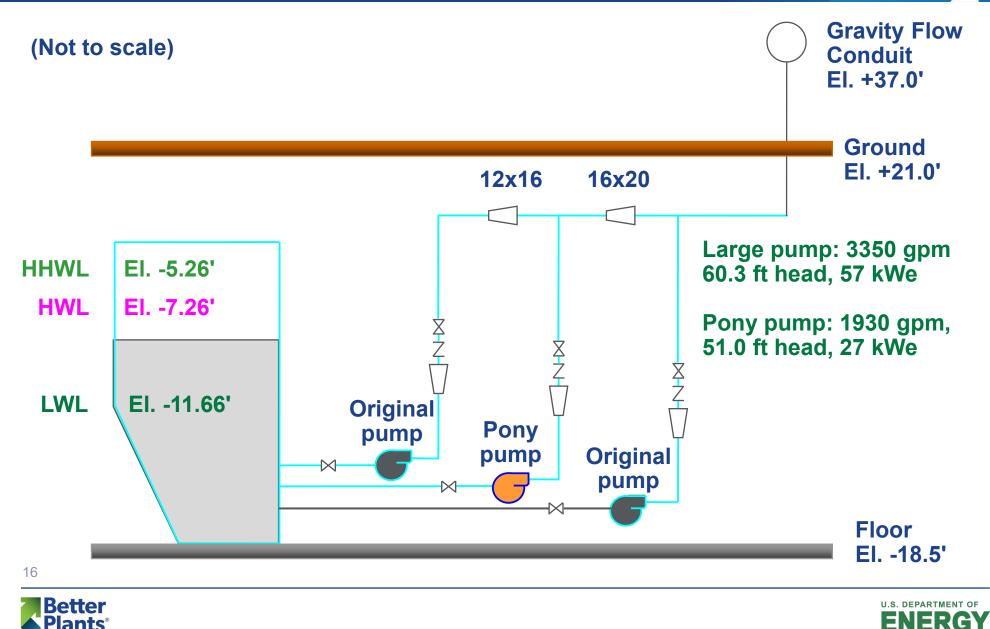
Savings = \$17,481 - \$13,234 = \$4,247/yr

| RESULTS | | SANKEY | HELP |
|-----------------------------|----------|--------|---------------------------|
| | Baseline | 0 | ptimized Pump at 1500 gpm |
| Percent Savings (%) | | | |
| Pump efficiency (%) | 82 | 8: | 2 |
| Motor rated power (hp) | 75 | 7 | 5 |
| Motor shaft power (hp) | 22.9 | 22 | 2.9 |
| Pump shaft power (hp) | 22.9 | 22 | 2.9 |
| Motor efficiency (%) | 88.3 | 8 | 8.3 |
| Motor power factor (%) | 57.8 | 5 | 7.8 |
| Percent Loaded (%) | 30 | 3(| D |
| Drive efficiency (%) | 100 | 1(| 00 |
| Motor current (amps) | 42 | 42 | 2 |
| Motor power (kW) | 19.3 | 19 | 9.3 |
| Annual Energy (MWh) | 161 | 1 | 61 |
| Annual Energy Savings (MWh) | _ | 0 | 0 |
| Annual Cost | \$13,234 | \$ | 13,234 |
| Annual Savings | _ | \$ | 00 |

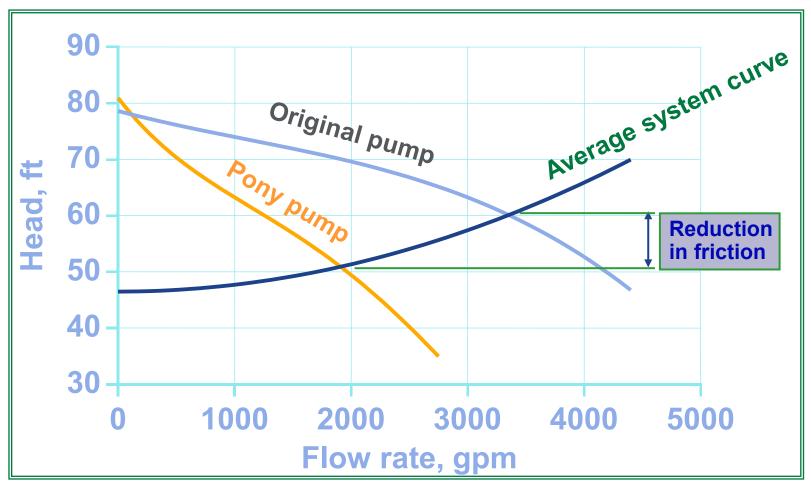




Lift station after replacing one large pump with smaller "pony" pump



The pony pump operates efficiently at lower flow rate, eliminating 2/3 of the frictional losses



Note: The sizing of the original pump, the availability of adequate spare capacity, and nature of the system made use of a variable speed drive less attractive for this particular system.





After making the design change:





Replacement pump comparison

BASELINE

| Pump Type | End Suction Sewage |
|--------------------|--------------------|
| ump Speed | 1170 |
| Drive | Direct Drive |
| luid Type | Water |
| luid Temperature | 68 |
| pecific Gravity | 1 |
| inematic Viscosity | 1 |
| itages | - + 1 |
| | |

| 60 Hz | ~ | |
|---------------------|-----|--|
| 75 | hp | |
| 1170 | rpm | |
| Standard Efficiency | ~ | |
| 460 | V | |
| 92.3 | А | |
| | | |

REPLACE PUMP

| Pump Efficiency | |
|-----------------|--|
| Optimize Pump | |

v

~

°F

cSt

rpm

| 79.6 | % |
|------|---|

The efficiency of your pump has been calculated based on your system setup. Either directly modify your efficiency or click "Optimize Pump" to estimate your pump efficiency based on a different pump type.

Pump Speed Drive Drive Efficiency Fluid Type Fluid Temperature Specific Gravity Kinematic Viscosity Stages Line Frequency Rated Motor Power Motor RPM Efficiency Class Efficiency Rated Voltage Full-Load Amps

Estimate Full-Load Amps

| 880 | rpm |
|----------------------|-----|
| Specified Efficiency | ~ |
| 100 | % |
| Water | ~ |
| 68 | °F |
| 1 | |
| 1 | cSt |
| - + 1 | |
| 60 Hz | ~ |
| 35 | hp |
| 880 | rpm |
| Specified | ~ |
| 86 | % |
| 460 | V |
| 49.2 | Α |



Line Frequency

Efficiency Class

Rated Voltage

Full-Load Amps

Motor RPM

Rated Motor Power



BASELINE

REPLACE PUMP

| Electricity Cost 0.082 SktWh Flow Rate 3350 gpm Head 60.3 T Catculate Head 60.3 T Load Estimation Method Power Implementation Costs Implementation Costs Motor Power 57 Implementation Costs Implementation Costs Implementation Costs Percent Savings (%) 18.0% Implementation Costs Implementation Costs Percent Savings (%) 72.3 79.6 Implementation Costs Implementation Costs Motor rated power (hp) 70.6 31.2 Imple | | | | | | | | |
|--|---------------------------|---------------------|---------------|----------|--------------------|--------------|-------|--------|
| Flow Rate 3350 gpm Flow Rate 1930 gpm Head Calculate Head 60.3 T Head 51 T Load Estimation Method Power Implementation Costs Impleme | Operating Hours | | 3741 | hrs/yr | Operating Hours | | 6482 | hrs/yr |
| Head Calculate Head Boo grm | Electricity Cost | | 0.082 | \$/kWh | Electricity Cost | | 0.082 | \$/kWh |
| Head Calculate Head 60.3 Itel Calculate Head 51 Itel Calculate Head Motor Power 57 WW Implementation Costs Implementation Costs | Flow Rate | | 3350 | gpm | Flow Rate | | 1930 | gpm |
| Motor Power F7 Implementation Costs Measured Voltage 600 V HE L P R E S U LT S S A N K E Y H E L P Percent Savings (%) 18.0% Pump efficiency (%) 72.3 79.6 Motor rated power (hp) 75 35 Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy (MWh) - 38 Annual Energy Savings (MWh) - 38 | | | 60.3 | | | | 51 | |
| Model okci SZ NV Measured Voltage 460 V R E S U LT S S A N K E Y H E L P Percent Savings (%) 18.0% Pump efficiency (%) 72.3 79.6 Motor rated power (hp) 75 35 Motor rated power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor refficiency (%) 92.4 86.3 Motor current (amps) 87 43 Motor current (amps) 87 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Energy Savings (MWh) - 38 | Load Estimation M | lethod | Power | ~ | | | | |
| RESULTS SANKEY HELP Baseline Replace Pump Percent Savings (%) 18.0% Pump efficiency (%) 72.3 79.6 Motor rated power (hp) 75 35 Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor reficiency (%) 92.4 86.3 Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) 38 Annual Cost \$17,485 \$14,338 | Motor Power | | 57 | kW | Implementation Cos | ts | | \$ |
| Baseline Replace Pump Percent Savings (%) Pump efficiency (%) 72.3 Pump efficiency (%) 72.3 Pump efficiency (%) 72.3 Pump efficiency (%) 72.3 Pump efficiency (%) 75 Motor rated power (hp) 75 Motor shaft power (hp) 70.6 Pump shaft power (hp) 70.6 Motor efficiency (%) 92.4 Motor efficiency (%) 82.2 Percent Loaded (%) 94 Porve efficiency (%) 100 Motor power (kW) 57 Motor power (kW) 57 Annual Energy (MWh) 213 Annual Energy Savings (MWh) - Annual Cost \$17,485 | Measured Voltage | | 460 | V | | | | |
| Baseline Replace Pump Percent Savings (%) Pump efficiency (%) 72.3 Motor rated power (hp) 75 Motor shaft power (hp) 70.6 Pump shaft power (hp) 70.6 Motor efficiency (%) 92.4 Motor efficiency (%) 82.2 Percent Loaded (%) 94 Porive efficiency (%) 100 Motor power (kW) 57 Motor power (kW) 57 Annual Energy (MWh) 213 Annual Energy Savings (MWh) Annual Cost \$17,485 | | PE | с III Т С | | SANKEY - | | HELD | |
| Percent Savings (%) 18.0% Pump efficiency (%) 72.3 Pump efficiency (%) 72.3 Motor rated power (hp) 75 Motor shaft power (hp) 70.6 Pump shaft power (hp) 70.6 Motor efficiency (%) 92.4 Motor power factor (%) 82.2 Percent Loaded (%) 94 Drive efficiency (%) 100 Motor power (kW) 57 Annual Energy (MWh) 213 Annual Energy Savings (MWh) - 38 Superartment of the state of the st | | RE | 30213 | | SANKET | | | |
| 20 18.0% 21 72.3 79.6 Motor rated power (hp) 75 35 Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor efficiency (%) 92.4 86.3 Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | | | Baseline | | Replace Pump | 0 | |
| Motor rated power (hp) 75 35 Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor efficiency (%) 92.4 86.3 Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | Percent Savings | (%) | | | | 18.0% | |
| Motor shaft power (hp) 70.6 31.2 Pump shaft power (hp) 70.6 31.2 Motor efficiency (%) 92.4 86.3 Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | Pump efficiency (| %) | 72.3 | | 79.6 | | _ |
| Pump shaft power (hp) 70.6 31.2 Motor efficiency (%) 92.4 86.3 Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | Motor rated powe | r (hp) | 75 | | 35 | | _ |
| Motor efficiency (%) 92.4 86.3 Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) – 38 Annual Cost \$17,485 \$14,338 | | Motor shaft powe | r (hp) | 70.6 | | 31.2 | | _ |
| Motor power factor (%) 82.2 78.3 Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor current (amps) 87 43 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | Pump shaft powe | r (hp) | 70.6 | | 31.2 | | _ |
| Percent Loaded (%) 94 89 Drive efficiency (%) 100 100 Motor current (amps) 87 43 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Percent Loaded (%) \$17,485 \$14,338 | | Motor efficiency (| %) | 92.4 | | 86.3 | | |
| Drive efficiency (%) 100 100 Motor current (amps) 87 43 Motor power (kW) 57 27 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) – 38 Annual Cost \$17,485 \$14,338 | | Motor power facto | or (%) | 82.2 | | 78.3 | | |
| 20 Motor current (amps) 87 43 20 Motor power (kW) 57 27 20 Annual Energy (MWh) 213 175 20 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | Percent Loaded (| %) | 94 | | 89 | | |
| 20 Motor power (kW) 57 27 20 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) - 38 Annual Cost \$17,485 \$14,338 | | Drive efficiency (% | %) | 100 | | 100 | | |
| 20 Annual Energy (MWh) 213 175 Annual Energy Savings (MWh) — 38 Annual Cost \$17,485 \$14,338 | | Motor current (an | nps) | 87 | | 43 | | |
| Annual Energy Savings (MWh) – 38 Annual Cost \$17,485 \$14,338 | | Motor power (kW |) | 57 | | 27 | | |
| Annual Energy Savings (MWh) – 38 Annual Cost \$17,485 \$14,338 | 20 | Annual Energy (| MWh) | 213 | | 175 | | |
| Better Plants* Us DEPARTMENT OF ENERGYAnnual Cost\$17,485\$14,338.s. DEPARTMENT OF ENERGYAnnual Savings—\$3,147 | | Annual Energy S | Savings (MWh) | _ | | 38 | | |
| LIS DEPARTMENT OF ENERGY Annual Savings — \$3.147 | Better | Annual Cost | | \$17,485 | | \$14,338 | | |
| **,*** | U.S. DEPARTMENT OF ENERGY | Annual Savings | | - | | \$3,147 | | |

What if you don't have room to ADD a pump?

- In this case, the excess redundancy allowed a pump to simply be replaced; in some systems, that option may not exist (e.g., space considerations)
- In such situations, a <u>properly selected</u> variable speed driven pump can provide nearly the same benefits (although with a higher capital cost) while maintaining required redundancy
- Replacing a functional pump may not be cost effective; but replacing a failed pump with a new design may





Y-12 Plant (Oak Ridge, TN) Demineralized water system

- This case study is based on work done by the Y-12 plant in Oak Ridge, Tennessee, a DOE defense facility.
- Case study technical contact:
- Don Casada—Qualified PSAT Instructor
- Diagnostic Solutions, LLC
- Email:doncasada@diagsol.com







Demineralized and tower water pumping station for the Fusion Energy complex







Let's investigate a really oversized pump – system operations have changed

Application:

Demineralized water pumps (for process cooling)

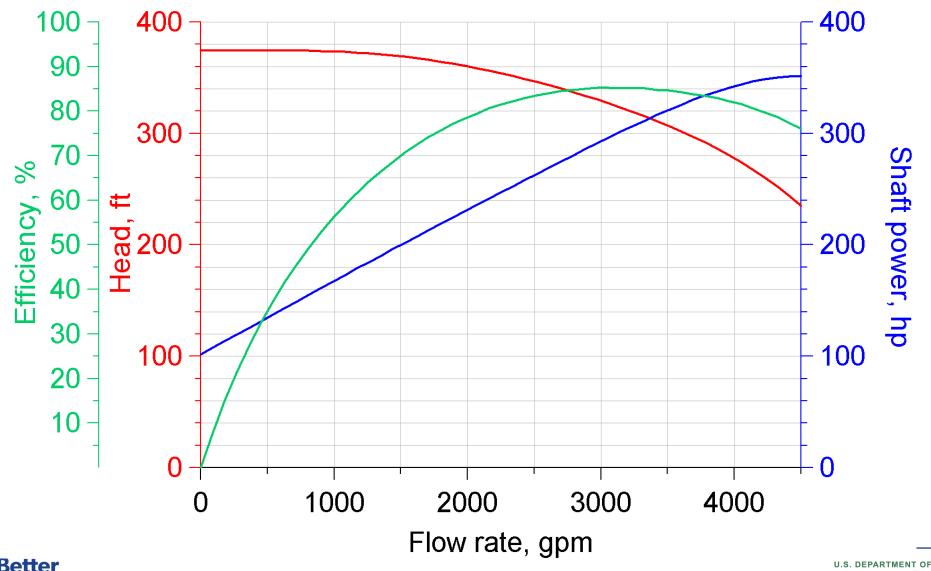
Original pump and motor design (4 parallel pumps): 3700 gpm @ 292 ft head, 1785 rpm pump 350 hp, 2300 V, 1785 rpm motor

Current system requirements: 1200 gpm @ 140 ft head (conservatively high)





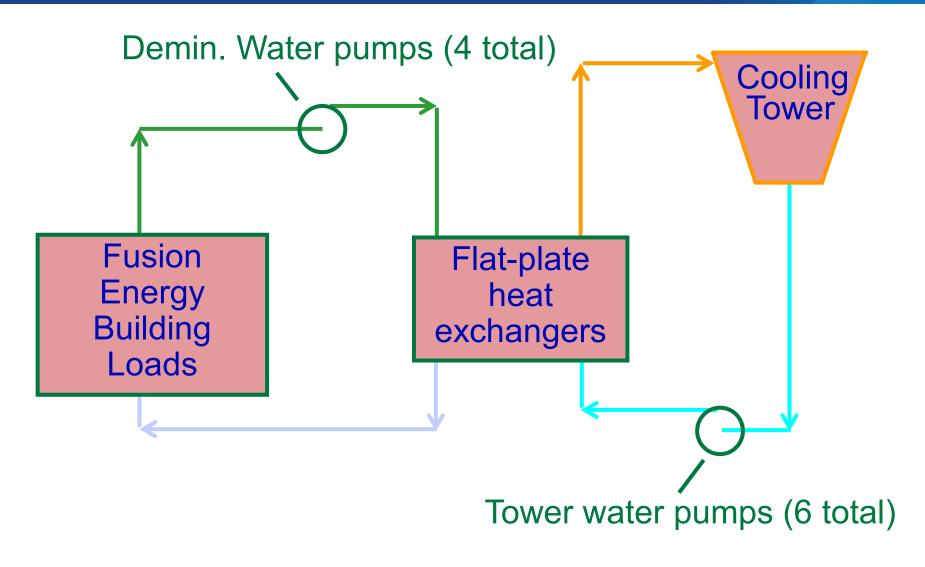
Installed pump performance curves



Bet



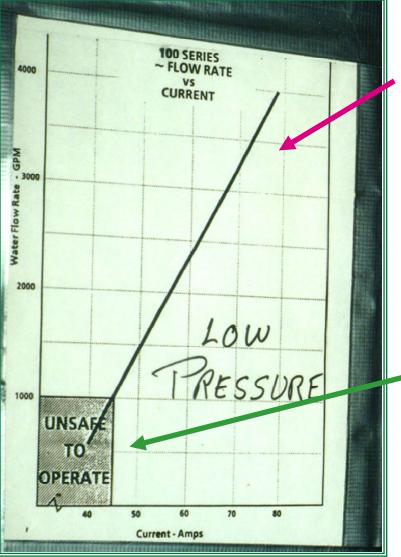
Simplified flow diagram







Operators can't always accommodate outdated engineering (i.e., changed facility demands).



Better

Sign on motor control center cabinet based on maintenance experience

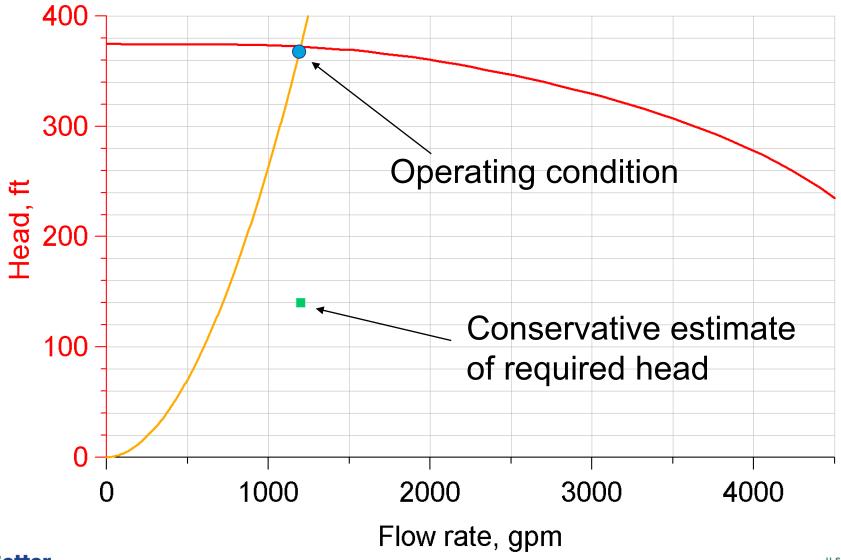
Ammeter on the same cabinet (typical operating condition)







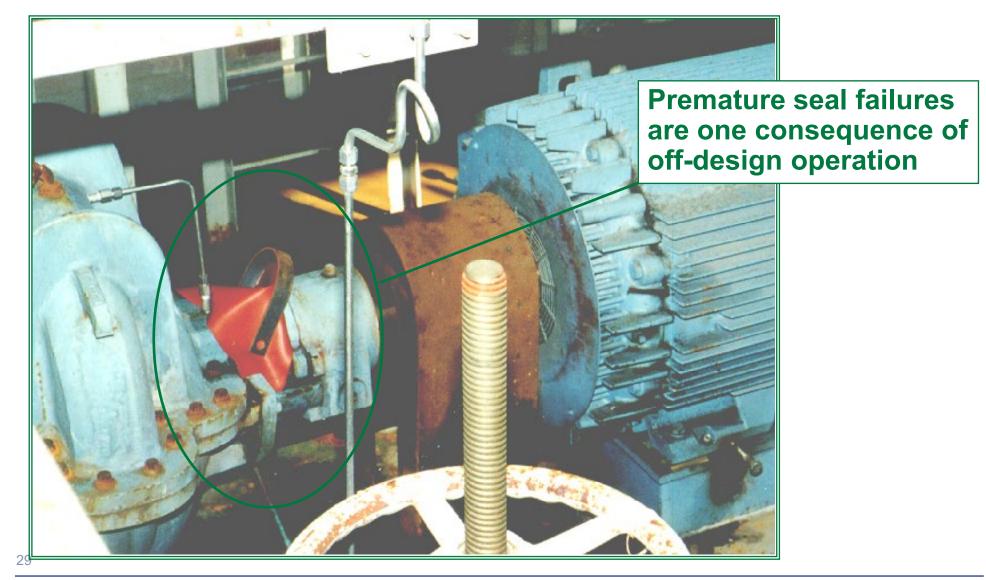
Even a conservative estimate clearly showed the effects of throttling/bypass losses







Off-design operation of pumps will result in increased operating AND maintenance costs







Applying the MEASUR tool to the measured conditions showed significant potential savings

| Pump Type | API Double Suction | ~ |
|---|---------------------|----|
| Pump Speed | 1785 rpn | n |
| Drive | Direct Drive | ~ |
| Fluid Type | Water | ~ |
| Fluid Temperature | 68 9 | F |
| Specific Gravity | 1 | |
| Kinematic Viscosity | 1 cs | st |
| Stages | - + 1 | |
| Line Frequency | 60 Hz | ~ |
| Rated Motor Power | 350 hj | р |
| Motor RPM | 1785 rpn | n |
| Efficiency Class | Standard Efficiency | ~ |
| Rated Voltage | 2300 | V |
| Full-Load Amps Estimate Full-Load Amps | 79 | Ą |

30





Applying the MEASUR tool to the measured conditions showed significant potential savings

| Operating Hours | 8760 | hrs/yr |
|------------------------|-------|--------|
| Electricity Cost | 0.054 | \$/kWh |
| Flow Rate | 1200 | gpm |
| Head Calculate Head | 367 | ft |
| Load Estimation Method | Power | ~ |
| Motor Power | 154 | kW |
| Measured Voltage | 2370 | V |





Applying the MEASUR tool to the measured conditions showed significant potential savings

| RESULTS | | SANKEY | | HELP |
|-----------------------------|----------|--------|--------|----------|
| | Baseline | | Optin | nal Pump |
| Percent Savings (%) | | | | 29.0% |
| Pump efficiency (%) | 57.3 | | 80.5 | |
| Motor rated power (hp) | 350 | | 350 | |
| Motor shaft power (hp) | 194.2 | | 138.2 |) |
| Pump shaft power (hp) | 194.2 | | 138.2 |) |
| Motor efficiency (%) | 94.1 | | 94.5 | |
| Motor power factor (%) | 80.1 | | 72.3 | |
| Percent Loaded (%) | 55 | | 39 | |
| Drive efficiency (%) | 100 | | 100 | |
| Motor current (amps) | 47 | | 37 | |
| Motor power (kW) | 154 | | 109.1 | |
| Annual Energy (MWh) | 1,349 | | 955 | |
| Annual Energy Savings (MWh) | _ | | 394 | |
| Annual Cost | \$72,848 | | \$51,5 | i83 |
| Annual Savings | _ | | \$21,2 | .65 |



Ultimate goal





Conservative estimate of required head – 140 Feet

BASELINE

Pump Type

Pump Speed

Drive

Fluid Type

Fluid Temperature

Specific Gravity

Kinematic Viscosity

Stages

| API Double Suction | ~ |
|--------------------|-----|
| 1785 | rpm |
| Direct Drive | ~ |
| Water | ~ |
| 68 | °F |
| 1 | |
| 1 | cSt |
| - + 1 | |

| Line Frequency |
|-------------------|
| Rated Motor Power |
| Motor RPM |
| Efficiency Class |
| Rated Voltage |
| Full-Load Amps |
| |

| 60 Hz | ~ |
|---------------------|-----|
| 350 | hp |
| 1785 | rpm |
| Standard Efficiency | ~ |
| 2300 | V |
| 79 | A |

OPTIMAL PUMP - 140 FEET HD

| Pump Type |
|------------------|
| Pump Efficiency |
| Known Efficiency |

API Double Suction

80.45 %

¥

The efficiency of your pump has been calculated based on your flow rate and selected pump type. Click "Known Efficiency" to use the efficiency calculated by your system setup.

| Pump Speed | 1785 |
|---|------|
| Drive | Spec |
| Drive Efficiency | 100 |
| Fluid Type | Wate |
| Fluid Temperature | 68 |
| Specific Gravity | 1 |
| Kinematic Viscosity | 1 |
| Stages | - + |
| Line Frequency | 60 H |
| Rated Motor Power | 75 |
| Motor RPM | 1785 |
| Efficiency Class | Stan |
| Rated Voltage | 2300 |
| Full-Load Amps Estimate Full-Load Amps | 17.7 |

| 1785 | rpm |
|----------------------|-----|
| Specified Efficiency | ~ |
| 100 | % |
| Water | ~ |
| 68 | °F |
| 1 | |
| 1 | cSt |
| - + 1 | |
| 60 Hz | ~ |
| 75 | hp |
| 1785 | rpm |
| Standard Efficiency | ~ |
| 2300 | V |
| 17.7 | А |





Conservative estimate of required head – 140 Feet

BASELINE

| Operating Hours | 8760 |
|------------------------|-------|
| Electricity Cost | 0.054 |
| Flow Rate | 1200 |
| Head Calculate Head | 367 |
| Load Estimation Method | Power |
| Motor Power | 154 |
| Measured Voltage | 2370 |

| 8760 | hrs/yi |
|-------|--------|
| 0.054 | \$/kWh |
| 1200 | gpm |
| 367 | ft |
| Power | ~ |
| 154 | kW |
| 2370 | V |

OPTIMAL PUMP - 140 FEET HD

| Operating Hours |
|------------------------|
| Electricity Cost |
| Flow Rate |
| Head Calculate Head |
| Implementation Costs |

| 8760 | I | hrs/yr |
|-------|----|--------|
| 0.054 | \$ | /kWh |
| 1200 | | gpm |
| 140 | | ft |

| 4 |
|---|
|---|





Conservative estimate of required head – 140 Feet

| RESULTS | SA | NKEY HELP |
|-----------------------------|----------|----------------------------|
| | Baseline | Optimal Pump - 140 Feet Hd |
| Percent Savings (%) | | 72.0% |
| Pump efficiency (%) | 57.3 | 80.5 |
| Motor rated power (hp) | 350 | 75 |
| Motor shaft power (hp) | 194.2 | 52.7 |
| Pump shaft power (hp) | 194.2 | 52.7 |
| Motor efficiency (%) | 94.1 | 92.2 |
| Motor power factor (%) | 80.1 | 81.6 |
| Percent Loaded (%) | 55 | 70 |
| Drive efficiency (%) | 100 | 100 |
| Motor current (amps) | 47 | 13 |
| Motor power (kW) | 154 | 42.6 |
| Annual Energy (MWh) | 1,349 | 373 |
| Annual Energy Savings (MWh) | _ | 976 |
| Annual Cost | \$72,848 | \$20,168 |
| Annual Savings | - | \$52,680 |





We considered some options

- Trim the pump impeller
- Get a new, smaller pump
- Add a variable speed drive

But what we finally decided was a little unconventional





A 125 hp, 6-pole (1190 rpm) motor was installed on an existing demineralized water pump



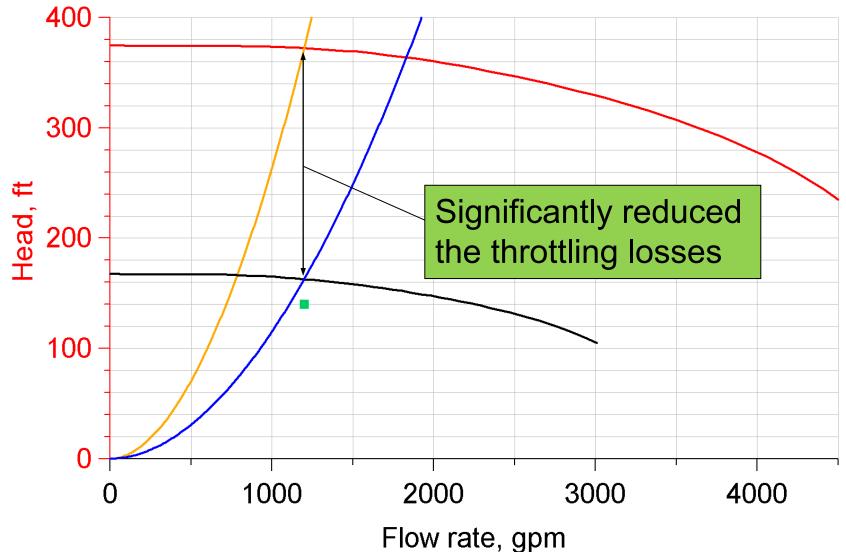
 $n_s = \frac{120 \text{ x Frequency}}{\text{No of Poles}}$

A motor with a broken foot was replaced





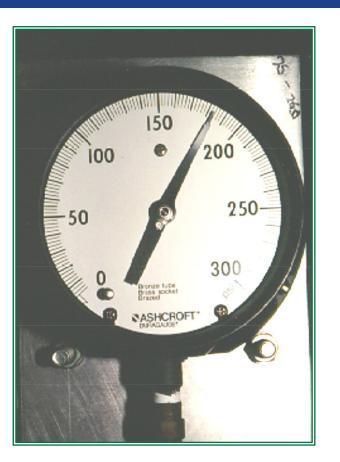
Operation of the pump at reduced speed eliminated much of the throttling losses







By slowing the motor down, the operating head was dramatically reduced, even at the same flow rate





Discharge gauges on identical parallel pumps; left gauge is for a pump driven by a 4-pole motor, right gauge is for the pump with a 6-pole motor. Note: suction is ~ 25 psig.





Before and After the Motor Change

BASELINE

| Pump Type | End Suction ANSI/AF | |
|---------------------|---------------------|--|
| Pump Speed | 1785 | |
| Drive | Direct Drive | |
| Fluid Type | Water | |
| Fluid Temperature | 68 | |
| Specific Gravity | 1 | |
| Kinematic Viscosity | 1 | |
| Stages | - + 1 | |

| ~ |
|-----|
| rpm |
| ~ |
| ~ |
| °F |
| |
| cSt |
| |
| |

Line Frequency Rated Motor Power Motor RPM Efficiency Class Rated Voltage Full-Load Amps

| ~ |
|-----|
| hp |
| rpm |
| ~ |
| V |
| А |
| |

REPLACE MOTOR (6-POLES)

Pump Efficiency Optimize Pump

Estimate Full-Load Amps

62.3

%

% ~

cSt

hp rpm

A

The efficiency of your pump has been calculated based on your system setup. Either directly modify your efficiency or click "Optimize Pump" to estimate your pump efficiency based on a different pump type.

| Pump Speed | 1190 |
|---------------------|----------------------|
| Drive | Specified Efficiency |
| Drive Efficiency | 100 |
| Fluid Type | Water |
| Fluid Temperature | 68 |
| Specific Gravity | 1 |
| Kinematic Viscosity | 1 |
| Stages | - + 1 |
| Line Frequency | 60 Hz |
| Rated Motor Power | 125 |
| Motor RPM | 1190 |
| Efficiency Class | Energy Efficient |
| Rated Voltage | 478 |
| Full-Load Amps | 142.72 |





Before and After the Motor Change

BASELINE

| Operating Hours | 8760 |
|------------------------|-------|
| Electricity Cost | 0.054 |
| Flow Rate | 1200 |
| Head | 367 |
| Calculate Head | |
| Load Estimation Method | Power |
| Motor Power | 154 |
| Measured Voltage | 2370 |

| 8760 | hrs/yr |
|-------|--------|
| 0.054 | \$/kWh |
| 1200 | gpm |
| 367 | ft |
| Power | ~ |
| 154 | kW |
| 2370 | V |

REPLACE MOTOR (6-POLES)

| Operating Hours | 8760 |
|----------------------|-------|
| Electricity Cost | 0.054 |
| Flow Rate | 1200 |
| Head | 162 |
| Calculate Head | |
| Implementation Costs | |

hrs/yr

\$/kWh

gpm

ft

\$





Before and After the Motor Change

| RESULTS | | SANKEY | HELP |
|-----------------------------|----------|--------|-------------------------|
| | Baseline | F | Replace Motor (6-Poles) |
| Percent Savings (%) | | | 60.0% |
| Pump efficiency (%) | 57.3 | 6 | 2.3 |
| Motor rated power (hp) | 350 | 1 | 25 |
| Motor shaft power (hp) | 194.2 | 7 | 8.8 |
| Pump shaft power (hp) | 194.2 | 7 | 8.8 |
| Motor efficiency (%) | 94.1 | g |)4.7 |
| Motor power factor (%) | 80.1 | - | 518.8 |
| Percent Loaded (%) | 55 | 6 | 3 |
| Drive efficiency (%) | 100 | 1 | 00 |
| Motor current (amps) | 47 | _ | 03 |
| Motor power (kW) | 154 | 6 | 2 |
| Annual Energy (MWh) | 1,349 | 5 | j43 |
| Annual Energy Savings (MWh) | _ | 8 | 06 |
| Annual Cost | \$72,848 | Ş | 29,349 |
| Annual Savings | _ | S | 43,500 |





Dollar and energy savings:

- Annual electricity cost reduction from this change are almost \$50,000 (other changes made to the system)
- Reduction in annual electrical energy is > 900,000 kWhr
- The motor/starter/cable capital cost was \$12,000
- Capital cost repaid in about 3 months





There were some important tangential benefits

- Seal face speed is reduced, seal life thereby extended
- Pump is more hydraulically stable, which means fewer maintenance problems are expected
- Noise levels are reduced both in the pump house and in the main Fusion building (hearing protection is no longer required)





Y-12 Plant: 9767-12 tower water pumps Multiple parallel pumps: A good idea.....

WHEN PROPERLY CONTROLLED

The temptation is to run more pumps than are needed, defeating the very reason for having multiple pumps.

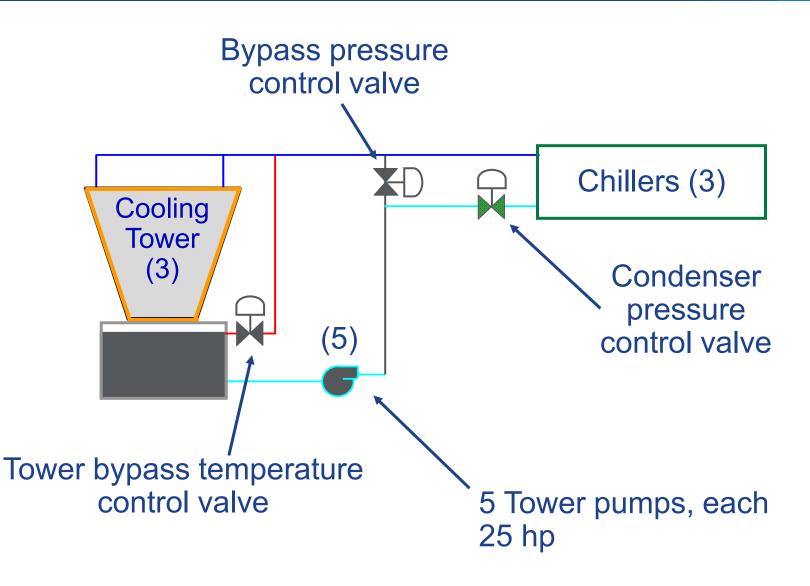


This case study is based on work done by the Y-12 plant in Oak Ridge, Tennessee, a DOE defense facility. Case study technical contact: Don Casada—Qualified PSAT Instructor Diagnostic Solutions, LLC Email:doncasada@diagsol.com





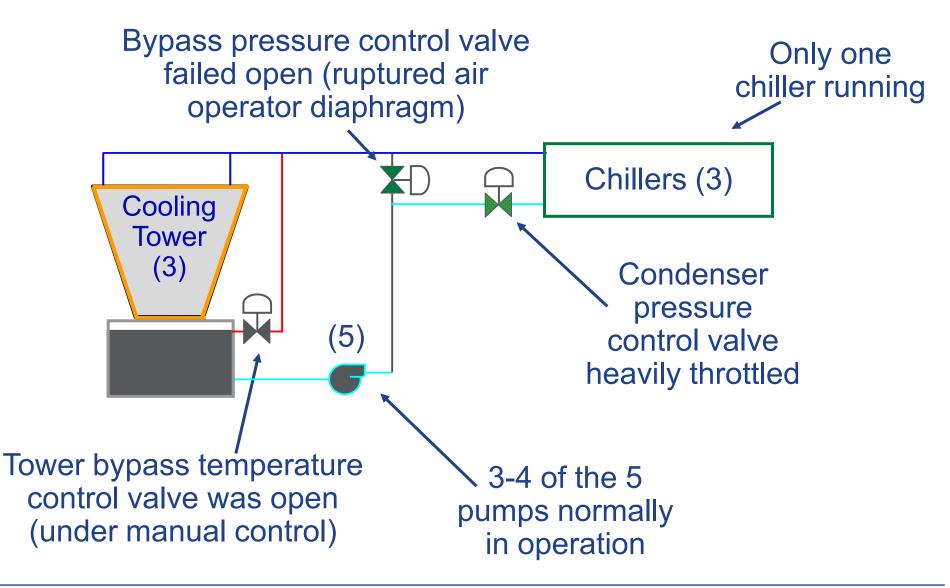
Simplified flow diagram of the tower water portion of the chilled water system







As found conditions: One chiller in operation, but 3 or 4 tower pumps running







Initial corrective actions were simple

- Closed manual isolation valve
- Repaired diaphragm in failed open bypass valve, eliminating bypass flow
- Turned off all but one or two tower pumps (depending on time of the year)
- Savings: about 30 kWe (\$14,000 per year)





A further look revealed additional energy reduction opportunities





Measured performance with only one original pump running (box around the pump & motor)

BASELINE

| Pump Type |
|-------------------|
| Pump Speed |
| Drive |
| Fluid Type |
| Fluid Temperature |
| Specific Gravity |

| Kinematic Viscosity |
|---------------------|
|---------------------|

Stages

| End Suction ANSI/API | ~ |
|----------------------|-----|
| 1750 | rpm |
| Direct Drive | ~ |
| Water | ~ |
| 68 | °F |
| 1 | |
| 1 | cSt |
| - + 2 | |

| Line Frequency |
|------------------|
| Rated Motor Powe |
| Motor RPM |
| Efficiency Class |
| Rated Voltage |
| |

Full-Load Amps



| 60 Hz | ~ |
|------------------|-----|
| 25 | hp |
| 1775 | rpm |
| Energy Efficient | ~ |
| 460 | V |
| 28.8 | А |

OPTIMAL PUMP

Motor RPM

Efficiency Class

Rated Voltage

Full-Load Amps

Estimate Full-Load Amps

| Pump Type | End Suction ANSI/API | ~ |
|--|----------------------|-----|
| Pump Efficiency Known Efficiency | 83.49 % | |
| The efficiency of your pump has been of selected pump type. Click "Known Effic | - | |
| Pump Speed | 1750 | rpm |
| Drive | Specified Efficiency | ~ |
| Drive Efficiency | 100 | % |
| Fluid Type | Water | ~ |
| Fluid Temperature | 68 | °F |
| Specific Gravity | 1 | |
| Kinematic Viscosity | 1 | cSt |
| Stages | - + 2 | |
| Line Frequency | 60 Hz | ~ |
| Rated Motor Power | 15 | hn |

✓
%
✓
°F
St ~ hp 15 1775 rpm Energy Efficient ¥ 460 V 18.39 А



Measured performance with only one original pump running (box around the pump & motor)

BASELINE

Operating Hours

Electricity Cost

Flow Rate

Head

Calculate Head

Load Estimation Method

Motor Current

Measured Voltage

| s/yr |
|------|
| Wh |
| pm |
| ft |
| |
| ~ |
| Α |
| V |
| |

OPTIMAL PUMP

Operating Hours

Electricity Cost

Flow Rate

Head

Calculate Head

Implementation Costs

| 8760 | hrs/yr |
|-------|--------|
| 0.054 | \$/kWh |
| 820 | gpm |
| 53.4 | ft |

| | \$ |
|--|----|
|--|----|





Measured performance with only one original pump running (box around the pump & motor)

| RESULTS | SAN | KEY | HELP |
|-----------------------------|--------------------------------|--------|----------|
| | Baseline | Optir | nal Pump |
| Percent Savings (%) | | | 35.0% |
| Pump efficiency (%) | 55.3 | 85.7 | |
| Motor rated power (hp) | 25 | 25 | |
| Motor shaft power (hp) | 20 | 12.9 | |
| Pump shaft power (hp) | 20 | 12.9 | |
| Motor efficiency (%) | 93.2 | 92.8 | |
| Motor power factor (%) | 84.1 | 71 | |
| Percent Loaded (%) | 80 | 52 | |
| Drive efficiency (%) | 100 | 100 | |
| Motor current (amps) | 23 | 18 | |
| Motor power (kW) | 16 | 10.4 | |
| Annual Energy (MWh) | 140 | 91 | |
| Annual Energy Savings (MWh) | — | 49 | |
| Annual Cost | \$7,572 | \$4,91 | 10 |
| Annual Savings | _ | \$2,66 | 32 |
| Ð | Potentia Savings ~\$2.7k | | U. |

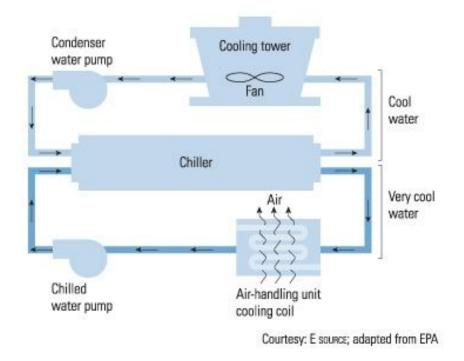
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ette

Stepping back to consider what is really required (the *Ultimate goal*)

A general rule of thumb for chillers: 3 gpm tower water flow per ton of cooling

(10° F rise in tower water for an 80% efficient chiller)







Reduce flow to 600 gpm @ 40 feet

BASELINE

| Pump Type | |
|------------|--|
| oump Speed | |
| Drive | |
| | |

Fluid Type

Fluid Temperature

Specific Gravity

Kinematic Viscosity

Stages

| Vertical Turbine | ~ |
|------------------|-----|
| 1750 | rpm |
| Direct Drive | ~ |
| Water | ~ |
| 68 | ۴ |
| 1 | |
| 1 | cSt |
| - + 2 | |

REDUCE FLOW TO 600 GPM

| Pump Type |
|-----------|
|-----------|

Vertical Turbine

85.66 %

Pump Efficiency Known Efficiency

The efficiency of your pump has been calculated based on your flow rate and selected pump type. Click "Known Efficiency" to use the efficiency calculated by your system setup.

| Pump Speed | 1750 | rpm |
|---------------------|----------------------|-----|
| Drive | Specified Efficiency | ~ |
| Drive Efficiency | 100 | % |
| Fluid Type | Water | ~ |
| Fluid Temperature | 68 | °F |
| Specific Gravity | 1 | |
| Kinematic Viscosity | 1 | cSt |
| Stages | - + 2 | |

REDUCE FLOW TO 600 GPM

BASELINE

| Line Frequency | 60 Hz |
|-------------------|-------|
| Rated Motor Power | 25 |
| Motor RPM | 1775 |
| Efficiency Class | Energ |
| Rated Voltage | 460 |
| Full-Load Amps | 28.8 |

| 60 Hz | ~ |
|------------------|-----|
| 25 | hp |
| 1775 | rpm |
| Energy Efficient | ~ |
| 460 | V |
| 28.8 | А |

| Line Frequency |
|-------------------|
| Rated Motor Power |
| Motor RPM |
| Efficiency Class |
| Rated Voltage |
| Full-Load Amps |

| 60 Hz | ~ |
|------------------|-----|
| 7.5 | hp |
| 1775 | rpm |
| Energy Efficient | ~ |
| 460 | V |
| 9.39 | А |

Reduce flow to 600 gpm @ 40 feet

BASELINE

Operating Hours

Electricity Cost

Flow Rate

Head

Calculate Head

Load Estimation Method

Motor Current

Measured Voltage

| 8760 | hrs/y |
|---------|--------|
| 0.054 | \$/kWh |
| 820 | gpm |
| 53.4 | ft |
| Current | ~ |
| 22.9 | A |
| 480 | V |
| | |

REDUCE FLOW TO 600 GPM

Operating Hours

Electricity Cost

Flow Rate

Head Calculate Head

Implementation Costs

| 8760 | hrs/yr |
|-------|--------|
| 0.054 | \$/kWh |
| 600 | gpm |
| 40 | ft |

| \$ | |
|----|--|





Reduce flow to 600 gpm @ 40 feet Replace with optimal pump

| RESULTS | | SANKEY | HELP |
|-----------------------------|----------|--------|------------------------|
| | Baseline | F | Reduce Flow to 600 GPM |
| Percent Savings (%) | | | 63.0% |
| Pump efficiency (%) | 55.3 | 8 | 85.7 |
| Motor rated power (hp) | 25 | (| 08 |
| Motor shaft power (hp) | 20 | (| 07.1 |
| Pump shaft power (hp) | 20 | (| 07.1 |
| Motor efficiency (%) | 93.2 | Ę | 90 |
| Motor power factor (%) | 84.1 | 8 | 32.3 |
| Percent Loaded (%) | 80 | Ę | 94 |
| Drive efficiency (%) | 100 | ŕ | 100 |
| Motor current (amps) | 23 | (| 09 |
| Motor power (kW) | 16 | (| 05.9 |
| Annual Energy (MWh) | 140 | (| 51 |
| Annual Energy Savings (MWh) | _ | 8 | 89 |
| Annual Cost | \$7,572 | (| \$2,773 |
| Annual Savings | - | ę | \$4,799 |





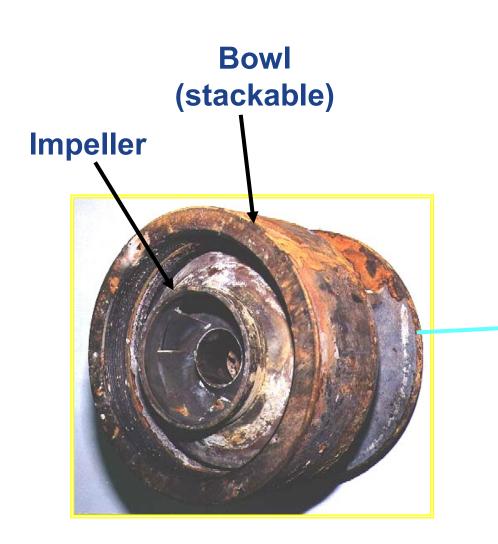
A great opportunity, but a familiar story....

NO CAPITAL FUNDS





Turning maintenance problems into energy savings....









Next slide:

Baseline: 2-stage pump (600 gpm @ 40 ft) Modification: 1-stage pump (600 gpm @ 45.6 ft) (saves about \$3,900/year)





Convert 2-stage pump to 1-stage pump

BASELINE

| Pump Type | End Suction ANSI/API |
|---------------------|----------------------|
| Pump Speed | 1750 |
| Drive | Direct Drive |
| Fluid Type | Water |
| Fluid Temperature | 68 |
| Specific Gravity | 1 |
| Kinematic Viscosity | 1 |
| Stages | - + 2 |

| End Suction ANSI/API | ~ |
|----------------------|-----|
| 1750 | rpm |
| Direct Drive | ~ |
| Water | ~ |
| 68 | °F |
| 1 | |
| 1 | cSt |
| - + 2 | |

1-STAGE PUMP

| Pump Efficiency |
|-----------------|
| Optimize Pump |

%

The efficiency of your pump has been calculated based on your system setup. Either directly modify your efficiency or click "Optimize Pump" to estimate your pump efficiency based on a different pump type.

73

| Pump Speed |
|---------------------|
| Drive |
| Drive Efficiency |
| Fluid Type |
| Fluid Temperature |
| Specific Gravity |
| Kinematic Viscosity |
| Stages |

| 1750 | rpm |
|----------------------|-----|
| Specified Efficiency | ~ |
| 100 | % |
| Water | ~ |
| 68 | °F |
| 1 | |
| 1 | cSt |
| - + 1 | |

BASELINE

| 60 Hz | ~ |
|------------------|---|
| 25 | hp |
| 1775 г | pm |
| Energy Efficient | ~ |
| 460 | V |
| 28.8 | Α |
| | 25 1775 r Energy Efficient 460 |

1-STAGE PUMP

| Line Frequency |
|---|
| Rated Motor Power |
| Motor RPM |
| Efficiency Class |
| Rated Voltage |
| Full-Load Amps Estimate Full-Load Amps |

| 60 Hz | ~ |
|------------------|-----|
| 25 | hp |
| 1775 | rpm |
| Energy Efficient | ~ |
| 460 | V |
| 28.8 | А |

ENERGY



Convert 2-stage pump to 1-stage pump

BASELINE

Operating Hours Electricity Cost

Flow Rate

Head

Calculate Head

Load Estimation Method

Motor Current

Measured Voltage

| 8760 | hrs/yr |
|---------|--------|
| 0.054 | \$/kWh |
| 600 | gpm |
| 40 | ft |
| | |
| Current | ~ |
| 22.9 | A |
| 480 | V |
| | |

1-STAGE PUMP

| Operating Hours |
|------------------|
| Electricity Cost |
| Flow Rate |
| Head |
| Calculate Head |

Implementation Costs

| 8760 | hrs/yr |
|-------|--------|
| 0.054 | \$/kWh |
| 600 | gpm |
| 45.6 | ft |

| | \$ |
|--|----|
|--|----|





Convert 2-stage pump to 1-stage pump

| RESULTS | | SANKEY | HELP |
|-----------------------------|----------|--------|--------------|
| | Baseline | | 1-Stage Pump |
| Percent Savings (%) | | | 52.0% |
| Pump efficiency (%) | 30.3 | | 73 |
| Motor rated power (hp) | 25 | | 25 |
| Motor shaft power (hp) | 20 | | 09.5 |
| Pump shaft power (hp) | 20 | | 09.5 |
| Motor efficiency (%) | 93.2 | | 91.7 |
| Motor power factor (%) | 84.1 | | 62.4 |
| Percent Loaded (%) | 80 | | 38 |
| Drive efficiency (%) | 100 | | 100 |
| Motor current (amps) | 23 | | 15 |
| Motor power (kW) | 16 | | 07.7 |
| Annual Energy (MWh) | 140 | | 67 |
| Annual Energy Savings (MWh) | _ | | 73 |
| Annual Cost | \$7,572 | | \$3,643 |
| Annual Savings | _ | | \$3,930 |







Pump system management

- Motor management programs have become relatively common
- Include repair/replace decision processes
- Opportunities with pumps are an order of magnitude greater than motors
- Consider a pump management program with contingency maintenance plans





The End for Case Studies





