

Industrial Fan Systems Virtual INPLT Training & Assessment

Session 2



MIIIII

Fan Virtual INPLT Facilitator



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- US Department of Energy, Advanced Manufacturing Office
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- United Nations Industrial Development Organization
- Air Movement and Control Association, AMCA International
 Many industrial clients both in the US and internationally



Agenda – Session 2

- Welcome and Introductions
- Safety and Housekeeping
- Agenda for Fan System Virtual INPLT (8 weeks)
- Today's Content:
 - **Industrial Fan Systems Fundamentals**
 - Fan and system curves
 - Fan types

MEASUR Tool

- Demonstration
- Kahoot Quiz Game
- Q&A



U.S. DEPARTMENT OF







Safety and Housekeeping

Safety Moment

- Make sure that belt guards are in place and firmly attached before removing the lockout/tagout and starting the fan
- You are welcome to ask questions at any time during the webinar
- When you are not asking a question, please <u>MUTE</u> your mic and this will provide the best sound quality for all participants
- We will be recording all these webinars and by staying on-line and attending the meeting you are giving your consent to be recorded
 - $\,\circ\,$ A link to the recorded webinars will be provided, afterwards





Fan system Virtual INPLT Agenda

- Week 1 Industrial Fan Systems Fundamentals and Introduction to MEASUR
- Week 2 Fan and system curves, Fan types
- Week 3 Fan affinity laws, Fan system controls
- Week 4 Creating a fan performance measurement plan & selecting measurement planes
- Week 5 Pressure considerations, Sizing ducts and estimating losses, Optimization techniques
- Week 6 Psychrometrics and air density for fan systems, System effect in fan systems
- Week 7 Fan system optimization strategies, Fan system evaluation with MEASUR
- Week 8 Industrial Fan System VINPLT Wrap-up Presentations





Learning objectives session 2

Class participants will:

- 1. Explain the general approach for developing a fan curve
- 2. Explain the interaction between the fan curve and the system curve
- 3. Understand the steps required to develop a system curve
- 4. Use pressure and flow data to develop a system curve





Fan and System Curves



"A graphical representation of how much flow a fan can develop over the range of fan static pressure that the fan can produce."







Fan Curve Development



- 1. Measure: flow, pressure, and power
- 2. Plot Data
- 3. Choke off Flow

Repeat Steps 1-3 until shut-off point is reached.







Centrifugal fan operation

- Fan imparts centrifugal force on the air
- Uses same principle as twirling a can of water
 - Fan spins
 - Air is hurled outward and held again fan housing
 - Air reaches fan outlet and flies outward, creating airflow







Fan Speed and the Fan Curve

- Fan speeds up: more flow and pressure
- Fan slows down: less flow and pressure















OPERATION AT STANDARD DENSITY













OPERATION AT STANDARD DENSITY





Fan Laws – More on this next week

- Also known as affinity laws
- Equations that relate fan speed to flow and pressure.
- Used to predict fan performance at different:
 - Fan speeds
 - Air densities (such as cold winter air or hot boiler gasses)
 - Size
 - Etc.
 - Etc.
 - Etc.





Combustion Air Fan - OSB

Combustion air blower







"A graphical representation of how much pressure is required to drive a certain amount of flow through the system."

What's a fan system?

- Everything attached to the fan, including:
 - Fume hoods
 - Ductwork
 - Volume control dampers
 - Filters
 - Heat exchangers
 - Driers





Types of devices that cause pressure drops in fan systems

- Static devices that don't change, like ductwork
- Changeable devices that can be set in a certain position, like dampers, or gradually get dirty, like filters
- Dynamic devices that change in response to a control signal





Static System Curve – Math Tricks

The curve is a parabola:

 $\Delta P \propto \Delta(Q)^2$

Oľ

 $Y = A \times X^2$

System Curve



What this means:

If we can measure the system pressure drop and flow rate at one point of operation, then we can calculate the pressure drop associated with other flow rates.





Changeable and Dynamic Devices

Airflow changes with time

Examples:

- Dampers are partially closed or opened to trim or increase flow
- Filters gradually accumulate dirt



System Curve





Combustion Air Fan OSB plant

- Dashed line:
 - Damper partially closed, pressure drop across orifice plate
- Solid line:
 - Damper open, orifice plate removed







Fan and System Curve Interaction

- The fan operates on the fan curve
- The system operates on the system curve
- Therefore, the operating point is...

The intersection of the fan curve and the system curve





Combustion Air Fan - OSB

Fan Curve and System Curve Interaction

Combustion air blower



- Point P:
 - Process requirements (8,080 cfm and 12 in. w.g.)
- Point 1:
 - Flow and pressure with open damper (16,000 cfm, 42 in. w.g. – assumes the motor doesn't overload)
- Point 2:
 - Flow and pressure with damper closed (8,080 cfm, 43 in. w.g.)





Variable Air Volume (VAV) Systems







VAV System Typical Characteristic Curve

Note: newer digital VAV systems have a very tiny differential pressure requirement across the VAV

Sometimes process systems have a fixed pressure requirement.



Air flow Q





Power Boiler – ID Fan has a fixed pressure component







VAV System Characteristic Curve and Fan Stall Regions for Backward Bladed Fans



Air flowrate Q





System curve with constant pressure and baghouse

$\mathbf{Y} = \mathbf{A} \times \mathbf{X}^2 + \mathbf{B} \times \mathbf{X} + \mathbf{C}$

This mathematical model of the system curve incorporates all the elements typically found in industrial fan systems:

- C the constant term represents the fixed pressure component
- B x X the linear term represents the loss across air cleaning devices such as a baghouse or a cooling coil
- $A \times X^2$ the square term represents the losses across ductwork

With a bit of cleverness when conducting a fan performance test, the constants can be calculated.





Can you think of any processes with a fixed pressure component?

Write your answer in the chat "to all"

No worries if someone else has already posted what you were planning to post, just post it anyway please.





MEASUR – Fan and System Curve Inputs

| MEASUR | | | \mathbf{k} | | | | | | - | |
|--------------------------|---------------------------------|------------------|---------------------|--------------|---------------------|-----------------------------------|-----------------------|----------------------------------|--|-------|
| IP fan assessment | | | | | Assessment D | iagram Report | t Sankey Calculat | ors | | |
| an Traverse | Analysis Optim | nal Fan Efficien | ncy Motor Pe | erformance N | EMA Energy Effi | ciency Fan Cu | irve | | | |
| Þ. | FAN CURVE | | | | | | GRAPH | | HELP | |
| Fan Cu | irve Data | | | | | Litt 2 | 1 | | | |
| | By Equation By Data | | | | | | | | | |
| Fan Data | Order | 4 | ~ | 4 | ~ | 25- | | | | |
| Flow | Order | Pressure | | Power | | Q 20 | | | and the second sec | |
| 0 | ft®/min | 19 | in H ₂ O | 40.8 | hp 🕄 | e (iu | | | Q | |
| 5000 | fts/min | 19.6 | in H₂O | 54.4 | hp 😆 | 02 20 Hui (in H20) 15 10 | | | | |
| 10100 | ft ^s /min | 19.9 | in H₂O | 66.6 | hp 😆 | <u>ک</u> 10 | | | | |
| 16500 | ft°/min | 20.3 | in H₂O | 83.8 | hp 😆 | 5 | | | | |
| 20900 | fts/min | 20.5 | in H₂O | 94.8 | hp 😆 | | | 2- 2- | 4. | |
| 26300 | ft®/min | 20.5 | in H ₂ O | 109 | hp 😫 | 0 | 10k- | 204 304 | 40k 50k | |
| 30200 | ft®/min | 19.9 | in H ₂ O | 116 | hp 😫 | | | | | x * * |
| 35200 | ft ^s /min | 18.3 | in H₂O | 122.6 | hp 😆 | 100 | | | | |
| 40200 | ft®/min | 15.5 | in H₂O | 123 | hp 😣 | | | | | |
| 45200 | ft ^s /min | 11.4 | in H₂O | 116.3 | hp 😆 | Power (hp) | | | | |
| 50900 | ft ^s /min | 5.6 | in H ₂ O | 100.6 | hp 😫 | Pod Sa | | | | |
| 55400 | ft ^s /min | 1.2 | in H ₂ O | 84.9 | hp 😆 | | | | | |
| | | | | | +Add Row | 0 | 10k- | 20k 30k | 40k 50k | |
| Baseline C | ondition | | | | | Lill 2 | | Flow (ft³/min) | | |
| Speed | ~ | | 1500 | | rpm | | | Flow (ft⁵/min) Pressure (in H₂O |) Fluid Power (hp.) | |
| | | | | | | | Baseline | | | |
| System Curve Data | | | | | - | Modification | | | | |
| System Cu | | - | | | | | System Curve | | | |
| Compressit System Los | bility Factor ss Exponent, C | L | 2 | | | | | | | |
| Point 1 | Liponom, o | Ŀ | ~ | | | Data Summ Key | ary Flow (ftª/mir | n) Pressure (in H₂O) | Efficiency (%) | Power |
| Flow Rate | | 1 | 0 | | ftª/min | ney | Flow (ft*/mir | ; Flessure (In H ₂ O) | Enciency (%) | (hp) |
| Pressure | | | 0 | | in H ₂ O | 0 | 40,165 | 15.39 | 79.7 | 121.9 |









Impeller Type Efficiency Worksheet

Application

- 1. Fabrication shop requiring 5,000 cfm ventilation air.
- 2. Process exhaust fan moving dust-laden air at 20,000 cfm, at 10 in. w.g. static pressure.
- 3. High-velocity fan used to chop paper trimmings and transport them to a central location.
- 4. 24,000 btu/hr. air conditioning unit required to deliver 800 cfm at 1.5 in. w.g. static pressure.
- 5. Industrial process requiring 50,000 cfm of clean air at 11 in. w.g. static pressure.
- 6. Tunnel ventilation application requiring 90,000 cfm at 3.5 in. w.g. static pressure.

Airfoil – Forward curved – Propeller

Backward-inclined – Vane Axial - Radial



Fan Type



Airfoil

Advantages

- Non-overloading
- Highest efficiency

Disadvantages

- High tip speed
- Hollow blades may fill wit



Applications

- High-velocity ventilation and supply systems

Optimization Tips

- Can be used in dirty settings if protected
- Requires careful setup














Backward-Curved

<u>Advantages</u>

- Good efficiency
- Can have nonstalling pressure/flow curve



<u>Applications</u>

- High-velocity ventilation and supply systems

Optimization Tips

- Can handle some dirt

Disadvantages

- High tip speed
- Underside of blades can fill with dust







Backward-Curved



DIDW









Backward-Inclined

Advantages

- Fairly good efficiency

Disadvantages

- Can stall at low flows
- Blades weaker than backward-curved blade



Applications

- High-velocity ventilation and supply systems
- Can be used in dirty settings if protected

Optimization Tips

- Can handle some dirt





Backward-Inclined (flat blade)









Radial Tipped

Advantages

- Strong Impeller
- Efficiency better than straight radial

Disadvantages

- Rising power characteristic
- Noisy
- Low efficiency

Applications

 Induced draught, high dust loadings

Optimization Tips

- Consider upgrading to BC or BI fan







Radial Shrouded

Advantages

- Very strong Impeller
- Simple construction

Disadvantages

- Rising power characteristic
- Noisy
- Low efficiency

Applications

Induced draught, high dust
 loadings

Optimization Tips

 Consider upgrading to radial tipped





Radial Shrouded







Axial Fan Types - Summary

Propeller



Vane Axial

(Includes guide vanes)



Tube Axial (no guide vanes)







Forward-Curved

Advantages

- Low tip peripheral speeds
- Quiet
- Compact

Disadvantages

- Severely rising power characteristic
- Weak impeller



Applications

- Ventilation

Optimization Tips

- Use for small HVAC applications only





Forward Curved







Centrifugal Fan Types Summary

Backward Curved (BCC)



– Backward Aerofoil (BAC)



- Backward Inclined (BIC)



Radial Bladed (RBC)



Forward Curved(FCC)



Radial Tipped (RT)







Centrifugal Impeller application

Decreasing impeller efficiency



Increasing dust loading





Propeller Fan

Advantages

- Lower first cost

Disadvantages

- Low efficiency
- Very low pressure

Applications

- Non-ducted

Optimization Tips

- Use for shop ventilation







Propeller Fans











Vane Axial Fan

Advantages

- Straight-through flow
- Negligible swirl, higher pressure than propeller & tube axial



Applications

- Low pressure
- Ducted
- <u>Optimization</u> <u>Tips</u>
- Use diffusers at inlet and outlet

Disadvantages

- High tip speed
- High sound level
- Low pressure developm







Polling questions 1 through 6

Polling Question

Fan Type

- 1) Fabrication shop requiring 5,000 cfm ventilation air.
- 2) Process exhaust fan moving dust-laden air at 20,000 cfm, at 10 in. w.g. static pressure.
- 3) High-velocity fan used to chop paper trimmings and transport them to a central location.
- 4) 24,000 btu/hr. air conditioning unit required to deliver 800 cfm at 1.5 in. w.g. static pressure.
- 5) Industrial process requiring 50,000 cfm of clean air at 11 in. w.g. static pressure.
- 6) Tunnel ventilation application requiring 90,000 cfm at 3.5 in. w.g. static pressure.
- a) Airfoil b) Forward curved c) Propeller
- d) Backward-inclined e) Vane Axial f) Radial



Application



Impeller Type Efficiency Worksheet

Application

- A. Fabrication shop requiring 5,000 cfm ventilation air.
- B. Process exhaust fan moving dust-laden air at 20,000 cfm, at 10 in. w.g. static pressure.
- C. High-velocity fan used to chop paper trimmings and transport them to a central location.
- D. 24,000 btu/hr. air conditioning unit required to deliver 800 cfm at 1.5 in. w.g. static pressure.
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Airfoil – Forward curved – Propeller Backward-inclined – Vane Axial - Radial



| Fan Type |
|----------|
| P |
| BI |
| R |
| FC |
| AF |
| VA |



Fans Choices in MEASUR

| | In MEASUR | Not in MEASUR |
|------------------|---|--|
| Centrifugal Fans | Airfoil Backward-Inclined Backward Curved Radial | Forward-CurvedIn-Line (Tubular centrifugal) |
| Axial Fans | Van Axial | Propeller Tube Axial |





Key Points / Action Items



- 1. The fan curve and system curve together can be used as a "road map" for navigating through a fan upgrade project, or for diagnosing system flow and pressure issues
- 2. Beware of selecting fans with their operating point too close to the surge point
- 3. Using the correct type of fan for the application is the first step towards a reliable and efficient system







- Find the fan curves for the top 5 fans from past week's list.
- For the top 10 fans from last week's list, determine what type of blade shape they have.
- Find a fan curve (and power curve if possible) for the fan associated with the top project candidate and enter the fan curve data in MEASUR





Thank You all for attending today's webinar.

See you all on next Thursday –

If you have specific questions, please stay online and we will try and answer them.

Alternately, you can email questions to me at ron@productiveenergy.com



Fan curves, power curves and efficiency curves





